

Turbomachinery Design Theory Solution

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Solution: Initial conditions: P 1 $\frac{1}{4}$ 7.5MPa, 5008C h 1 $\frac{1}{4}$ 3404.3kJ/kg s 1 $\frac{1}{4}$ 6.7598kJ/kg K (h 1 and s 1 from superheated steam tables) At the exit state, P 2 . P c $\frac{1}{4}$ 00:545P67:5P $\frac{1}{4}$ 4:0875MPa; and therefore the nozzle is convergent. State 2 is fixed by P 2 $\frac{1}{4}$ 5MPa,s 1 $\frac{1}{4}$ s 2 $\frac{1}{4}$ 6.7598kJ/kgK T 2 $\frac{1}{4}$ 4358K, v 2 $\frac{1}{4}$ 0.06152m³/kg, h 2 $\frac{1}{4}$ 3277 ...

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We usually assume a single axisymmetric S2 surface and several untwisted S1 surfaces. The S2 (hub to tip or throughflow) solution has become the “backbone” of turbomachinery design. Initially there was rivalry between the matrix-stream function method and the streamline curvature method of solving the equations.

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